

Harmonization and improvements to the test procedure for Fast Cook-Off (STANAG 4240)



Group's goals:

- ✓ Harmonize Fast Heating test procedures and acceptance criteria
- ✓ Look for new alternative solutions
- ✓ Evaluate equivalence of liquid fuel and alternative solutions

POOL FIRE TESTS – How standardized?

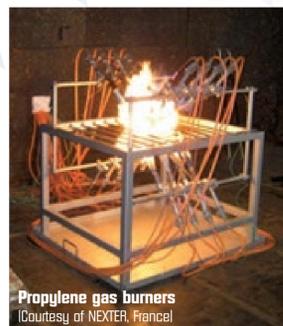
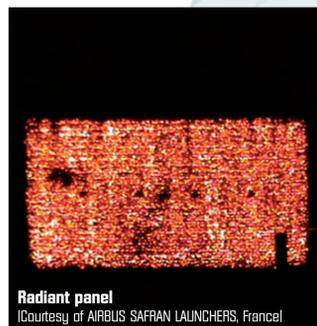
- ✓ Repeatability and uniformity are difficult to ensure
- ✓ Wind is identified as a critical test parameter



ALTERNATIVE SETUPS - An opportunity to improve test repeatability?

- ✓ The next standard for Fast Heating testing will include the possibility to perform alternative aggressions provided that they are proved to be representative of fuel fires

State of the art – Liquefied Propane Gas burners, propane jets, sand-bed burners, radiant panels,...



- ✓ Very different alternative test setups experimented with throughout the world: WTD91 (Germany), NSWC (US), NAWCDDW (US), BTC (Sweden), TNO (Netherlands), AIRBUS SAFRAN LAUNCHERS (France), NEXTER (France), DIEHL BGT Defence (Germany), Hirtenberger DS (Austria), etc.
- ✓ Large or small scale facilities (for representativity or research purpose)

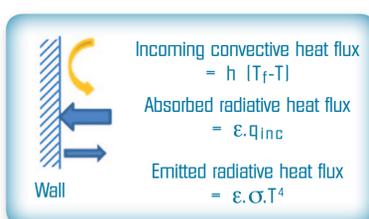
The consistency of IM test results shall be guaranteed whatever the fire test method

FUTURE CHALLENGES – IMEMG companies views:

- ✓ EQUIVALENCE between Liquid Fuel Fires and alternative setups :
 - Comparable heat flux data achieved on some tests BUT dispersions observed in both cases
 - Equivalence needs to be validated for all phases (ignition and hot phases)
- ✓ CALIBRATION DEVICES AND CRITERIA :
 - Harmonized apparatus necessary to obtain comparable measurements

Which heat flux?

Assuming the net wall heat flux density may be expressed by the following equation, a comparison of heat flux outputs expected by existing fire heat flux meters is done in the hereafter table: $\Phi_{net} = h \cdot (T_f - T) + \epsilon \cdot (q_{inc} - \sigma \cdot T^4)$



Device	Expected output	Comment
Adiabatic Surface Temperature method (AST probe, PT, ...)	$q_{inc} = \sigma \cdot T_{AST}^4 \cdot h / \epsilon \cdot (T_f - T_{AST})$	Incident radiative heat flux at equilibrium, $\Phi_{net} = 0 \Rightarrow T = T_{AST}$
Directional Slug Calorimeter	$\Phi_{DSC} = h \cdot (T_f - T) + \epsilon \cdot q_{inc}$	Total absorbed heat flux
Bi-thermocouples	q_{inc}	Incident radiative heat flux
Calorimeter, Directional Flame Thermometer	Φ_{net}	Total net heat flux

T = wall temperature, T_f = gas temperature, h = convective heat transfer coefficient, ϵ = wall emissivity, q_{inc} = incident radiative heat flux

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